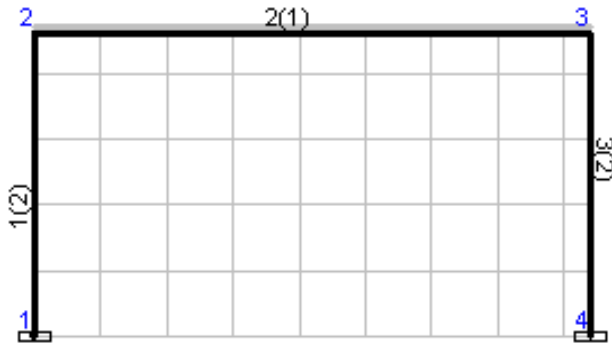


**1-Finite element model (FEM)**



**Nodal points**

Node	x [m]	y [m]
1	0.000	0.000
2	0.000	4.600
3	8.400	4.600
4	8.400	0.000

**Supports**

Node	kind	ux [mm]	uy [mm]	ur [rad]
1	fixed	ux=uy=ur=0		
4	fixed	ux=uy=ur=0		

**Materials**

Material : Steel, E= 210.000 [GPa]  
 Weight density :  $\rho = 78.500$  [kN/m<sup>3</sup>]  
 The element self weight is included in loads and masses

**Element cross sections**

Cr.sec.	b [mm]	h [mm]	Ac [mm <sup>2</sup> ]	Ic [mm <sup>4</sup> ]
1			1.15500E+004	4.82000E+008
2			1.56000E+004	9.20800E+008

**Elements**

Element	node 1	node 2	material	length(m)	angle(°)
1	1	2	2	4.600	90.000
2	2	3	1	8.400	0.000
3	3	4	2	4.600	270.000

**Nodal loads vertical forces Fy, (yg=1.35, yq=1.50)**

Node	Fyg [kN]	Fyq [kN]	ygFyg+yqFyq [kN]
2	-95.000	-125.000	-315.750
3	-95.000	-125.000	-315.750

## Designing frame structures from Steel

### Element distributed loads ( $\gamma_g=1.35$ , $\gamma_q=1.50$ )

element	G[kN/m]	Q[kN/m]	$\gamma_g G + \gamma_q Q$ [kN/m]	load kind	load direction
2	8.600	12.400	30.210	uniform	perpendicular

### Element distributed loads due to self weight ( $\gamma_g=1.35$ , $\gamma_q=1.50$ )

element	G[kN/m]	Q[kN/m]	$\gamma_g G + \gamma_q Q$ [kN/m]	load kind	load direction
1	1.225	0.000	1.654	uniform	vertical
2	0.907	0.000	1.224	uniform	vertical
3	1.225	0.000	1.654	uniform	vertical

## 2-Results of static-linear-elastic analysis

### Diagrams of internal forces M, V, N, and displacements d, of element 1

n	x/l	x[m]	M[kNm]	V[kN]	N[kN]	dx[mm]	dy[mm]	d[mm]
0	0.000	0.00	78.05	51.98	-455.38	0.000	0.000	0.000
1	0.100	0.46	54.14	51.98	-454.62	-0.038	-0.063	0.074
2	0.200	0.92	30.23	51.98	-453.86	-0.136	-0.127	0.186
3	0.300	1.38	6.32	51.98	-453.10	-0.267	-0.190	0.327
4	0.400	1.84	-17.59	51.98	-452.34	-0.404	-0.254	0.477
5	0.500	2.30	-41.50	51.98	-451.58	-0.523	-0.317	0.611
6	0.600	2.76	-65.41	51.98	-450.82	-0.596	-0.380	0.707
7	0.700	3.22	-89.32	51.98	-450.06	-0.597	-0.444	0.744
8	0.800	3.68	-113.23	51.98	-449.30	-0.501	-0.507	0.713
9	0.900	4.14	-137.14	51.98	-448.54	-0.280	-0.571	0.636
10	1.000	4.60	-161.05	51.98	-447.77	0.090	-0.634	0.641

#### Maximum values for element 1

maxM=	78.05 kNm,	minM=	-161.05 kNm
maxV=	51.98 kN,	minV=	51.98 kN
maxN=	-447.77 kN,	minN=	-455.38 kN
maxd=	0.744 mm		

### Diagrams of internal forces M, V, N, and displacements d, of element 2

n	x/l	x[m]	M[kNm]	V[kN]	N[kN]	dx[mm]	dy[mm]	d[mm]
0	0.000	0.00	-161.05	-132.02	-51.98	0.090	-0.634	0.641
1	0.100	0.84	-61.23	-105.62	-51.98	0.072	-1.902	1.903
2	0.200	1.68	16.40	-79.21	-51.98	0.054	-3.610	3.611
3	0.300	2.52	71.85	-52.81	-51.98	0.036	-5.217	5.217
4	0.400	3.36	105.12	-26.40	-51.98	0.018	-6.335	6.335
5	0.500	4.20	116.21	0.00	-51.98	0.000	-6.733	6.733
6	0.600	5.04	105.12	26.40	-51.98	-0.018	-6.335	6.335
7	0.700	5.88	71.85	52.81	-51.98	-0.036	-5.217	5.217
8	0.800	6.72	16.40	79.21	-51.98	-0.054	-3.610	3.611
9	0.900	7.56	-61.23	105.62	-51.98	-0.072	-1.902	1.903
10	1.000	8.40	-161.04	132.02	-51.98	-0.090	-0.634	0.640

#### Maximum values for element 2

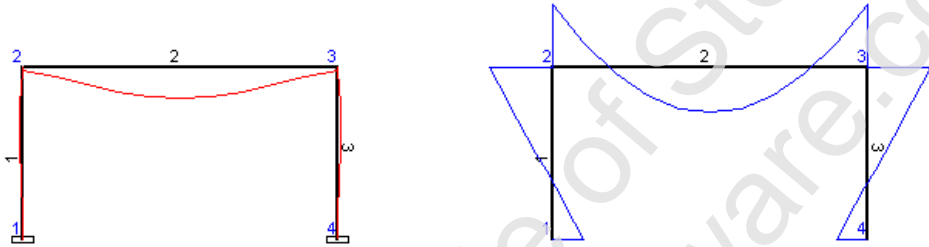
maxM=	116.21 kNm,	minM=	-161.05 kNm
maxV=	132.02 kN,	minV=	-132.02 kN
maxN=	-51.98 kN,	minN=	-51.98 kN
maxd=	6.733 mm		

**Diagrams of internal forces M, V, N, and displacements d, of element 3**

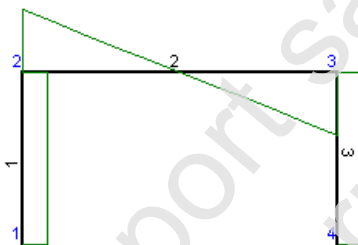
n	x/l	x [m]	M [kNm]	V [kN]	N [kN]	dx [mm]	dy [mm]	d [mm]
0	0.000	0.00	-161.05	-51.98	-447.77	-0.090	-0.634	0.640
1	0.100	0.46	-137.14	-51.98	-448.54	0.280	-0.571	0.636
2	0.200	0.92	-113.23	-51.98	-449.30	0.501	-0.507	0.713
3	0.300	1.38	-89.32	-51.98	-450.06	0.597	-0.444	0.744
4	0.400	1.84	-65.41	-51.98	-450.82	0.595	-0.380	0.706
5	0.500	2.30	-41.50	-51.98	-451.58	0.523	-0.317	0.611
6	0.600	2.76	-17.59	-51.98	-452.34	0.404	-0.254	0.477
7	0.700	3.22	6.32	-51.98	-453.10	0.267	-0.190	0.328
8	0.800	3.68	30.23	-51.98	-453.86	0.136	-0.127	0.186
9	0.900	4.14	54.14	-51.98	-454.62	0.038	-0.063	0.074
10	1.000	4.60	78.05	-51.98	-455.38	0.000	0.000	0.000

**Maximum values for element 3**

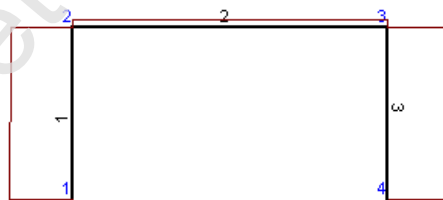
maxM= 78.05 kNm, minM= -161.05 kNm  
 maxV= -51.98 kN, minV= -51.98 kN  
 maxN= -447.77 kN, minN= -455.38 kN  
 maxd= 0.744 mm



Displacement diagram  
maxD=6.73 mm



Bending moment diagram  
maxM=115.21 kNm, minM=-161.05 kNm



Shear force diagram  
maxV=132.02 kN, minV=-132.02 kN

Axial force diagram  
maxN=-51.98 kN, minN=-455.38 kN

**3-Dimensioning of Steel (EC3 EN1993-1-1:2005)**

**Design codes**

EN1990:2002, Eurocode 0 Basis of Structural Design  
 EN1991-1-1:2002, Eurocode 1-1 Actions on structures  
 EN1993-1-1:2005, Eurocode 3 Steel structures  
 EN1997-1-1:2004, Eurocode 7 Geotechnical design  
 EN1998-1-1:2004, Eurocode 8 Design in earthquake environment  
 NA - National Annex:

**Materials**

**Steel: S 355** (EN1993-1-1, §3.2)

$t \leq 40$  mm, Yield strength  $f_y = 355$  N/mm<sup>2</sup>, Ultimate strength  $f_u = 510$  N/mm<sup>2</sup>  
 $40\text{mm} < t \leq 80$  mm, Yield strength  $f_y = 335$  N/mm<sup>2</sup>, Ultimate strength  $f_u = 470$  N/mm<sup>2</sup>  
 Modulus of elasticity  $E = 210000$  N/mm<sup>2</sup>, Poisson ratio  $\nu = 0.30$ , Unit mass  $\rho = 7850$  Kg/m<sup>3</sup>

**Partial safety factors for actions** (EN1990, Annex A1)

$\gamma_G = 1.35$ ,  $\gamma_Q = 1.50$ ,  $\psi = 0.30$

**Partial factors for materials** (EN1993-1-1, §6.1)

$\gamma_{M0} = 1.00$ ,  $\gamma_{M1} = 1.00$ ,  $\gamma_{M2} = 1.25$

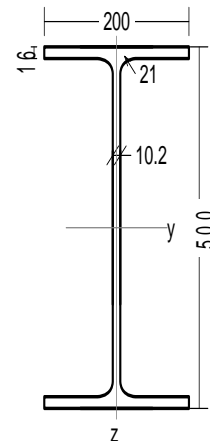
**Cross-section properties**

Cross-section : 1, IPE 500-S 355

**Dimensions of cross section**

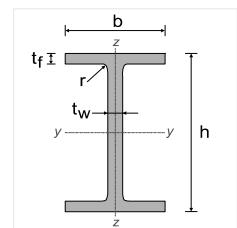
Depth of cross section	$h =$	500.00 mm
Width of cross section	$b =$	200.00 mm
Web depth	$h_w =$	468.00 mm
Depth of straight portion of web	$d_w =$	426.00 mm
Web thickness	$t_w =$	10.20 mm
Flange thickness	$t_f =$	16.00 mm
Radius of root fillet	$r =$	21.00 mm
Mass	$=$	90.70 Kg/m

IPE 500



**Properties of cross section**

Area	$A =$	11550 mm <sup>2</sup>	
Second moment of area	$I_y = 482.00 \times 10^6$	mm <sup>4</sup>	$I_z = 21.420 \times 10^6$ mm <sup>4</sup>
Section modulus	$W_y = 1928.0 \times 10^3$	mm <sup>3</sup>	$W_z = 214.20 \times 10^3$ mm <sup>3</sup>
Plastic section modulus	$W_{py} = 2194.0 \times 10^3$	mm <sup>3</sup>	$W_{pz} = 335.90 \times 10^3$ mm <sup>3</sup>
Radius of gyration	$i_y =$	204.3 mm	$i_z =$ 43.1 mm
Shear area	$A_{vz} =$	5985 mm <sup>2</sup>	$A_{vy} =$ 6718 mm <sup>2</sup>
Torsional constant	$I_t = 0.893 \times 10^6$	mm <sup>4</sup>	$i_p =$ 209 mm
Warping constant	$I_w = 1249.4 \times 10^9$	mm <sup>6</sup>	

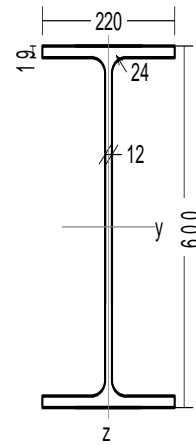


Cross-section : 2, IPE 600-S 355

**Dimensions of cross section**

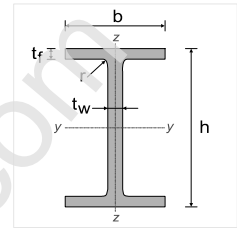
Depth of cross section	h=	600.00 mm
Width of cross section	b=	220.00 mm
Web depth	hw=	562.00 mm
Depth of straight portion of web	dw=	514.00 mm
Web thickness	tw=	12.00 mm
Flange thickness	tf=	19.00 mm
Radius of root fillet	r=	24.00 mm
Mass	=	122.00 Kg/m

IPE 600



**Properties of cross section**

Area	A=	15600 mm <sup>2</sup>	
Second moment of area	I <sub>y</sub> =	920.80x10 <sup>6</sup> mm <sup>4</sup>	I <sub>z</sub> =33.870x10 <sup>6</sup> mm <sup>4</sup>
Section modulus	W <sub>y</sub> =	3069.0x10 <sup>3</sup> mm <sup>3</sup>	W <sub>z</sub> =307.90x10 <sup>3</sup> mm <sup>3</sup>
Plastic section modulus	W <sub>py</sub> =	3512.0x10 <sup>3</sup> mm <sup>3</sup>	W <sub>pz</sub> =485.60x10 <sup>3</sup> mm <sup>3</sup>
Radius of gyration	i <sub>y</sub> =	243.0 mm	i <sub>z</sub> = 46.6 mm
Shear area	A <sub>vz</sub> =	8380 mm <sup>2</sup>	A <sub>vy</sub> = 8792 mm <sup>2</sup>
Torsional constant	I <sub>t</sub> =	1.654x10 <sup>6</sup> mm <sup>4</sup>	i <sub>p</sub> = 247 mm
Warping constant	I <sub>w</sub> =	2845.5x10 <sup>9</sup> mm <sup>6</sup>	



**Steel design, element 1 , L= 4.600m, IPE 600**

**Classification of cross-sections, Bending and compression**

(EN1993-1-1, §5.5)

Maximum and minimum cross-section stresses  $\sigma = N_{ed}/A_{el} \pm M_{y,ed}/W_{el,y} \pm M_{z,ed}/W_{el,z}$   
 $\sigma_1 = 82 \text{ N/mm}^2$ ,  $\sigma_2 = -23 \text{ N/mm}^2$  (compression positive)

Web

$c = 600.0 - 2 \times 19.0 - 2 \times 24.0 = 514.0 \text{ mm}$ ,  $t = 12.0 \text{ mm}$ ,  $c/t = 514.0/12.0 = 42.83$   
 S 355 ,  $t = 12.0 \leq 40 \text{ mm}$ ,  $f_y = 355 \text{ N/mm}^2$ ,  $\epsilon = (235/355)^{0.5} = 0.81$   
 Position of neutral axis for combined Bending and compression  
 $N_{ed}/(2t_w \cdot f_y/\gamma_{M0}) = 455382/(2 \times 12.0 \times 355/1.00) = 53.4 \text{ mm}$   
 $\alpha = (514.0/2 + 53.4)/514.0 = 0.604 > 0.5$   
 $c/t = 42.83 \leq 396 \times 0.81 / (13 \times 0.604 - 1) = 46.81$   
 The web is class 1 (EN1993-1-1, Tab.5.2)

Flange

$c = 220.0/2 - 12.0/2 - 24.0 = 80.0 \text{ mm}$ ,  $t = 19.0 \text{ mm}$ ,  $c/t = 80.0/19.0 = 4.21$   
 S 355 ,  $t = 19.0 \leq 40 \text{ mm}$ ,  $f_y = 355 \text{ N/mm}^2$ ,  $\epsilon = (235/355)^{0.5} = 0.81$   
 $c/t = 4.21 \leq 9 \times 0.81 = 7.29$   
 The flange is class 1 (EN1993-1-1, Tab.5.2)

**Overall classification of cross-section is Class 1, Bending and compression  $N_{c,ed} + M_{y,ed}$**

**Resistance of cross-section**

(EN1993-1-1 §6.2.3, §6.2.4, §6.2.6, §6.2.5)

Tension Resistance	$N_{rdt,rd} = A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 15600 \times 355 / 1.00 = 5538.00 \text{ kN}$
Compression Resistance	$N_{rdc,rd} = A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 15600 \times 355 / 1.00 = 5538.00 \text{ kN}$
Shear resistance	$V_{rdz,rd} = A_v (f_y / \sqrt{3}) / \gamma_{M0} = [10^{-3}] \times 8380 \times (355 / 1.73) / 1.00 = 1717.56 \text{ kN}$
Bending Resistance	$M_{rdy,rd} = W_{ely} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 3512.0 \times 10^3 \times 355 / 1.00 = 1246.76 \text{ kNm}$

## Designing frame structures from Steel

### Steel design, element 1, [Span ], L= 4.600m, IPE 600

Med = -89.32 kNm, Ved = 51.98 kN, Ned = -450.06 kN

#### Ultimate Limit State , Verification for compression (EN1993-1-1, §6.2.4)

Nc.ed=450.06 kN

Compression Resistance  $N_{plrd}=A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 15600 \times 355 / 1.00 = 5538.00 \text{ kN}$   
Ned= 450.06 kN < 5538.00 kN =Nc,rd=Nplrd, Check is verified

#### Ultimate Limit State , Verification for bending moment y-y (EN1993-1-1, §6.2.5)

My.ed= 89.32 kNm

Bending Resistance  $M_{ply,rd}=W_{ply} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 3512.0 \times 10^3 \times 355 / 1.00 = 1246.76 \text{ kNm}$   
My,ed= 89.32 kNm < 1246.76 kNm =My,rd=Mply,rd, Check is verified

#### Ultimate Limit State , Verification for axial force, shear and bending (EN1993-1-1, §6.2.9)

N.ed= 450.06kN (Compression), Vz.ed= 51.98kNm, My.ed= 89.32kN

Nplrd=5538.00kN, Mpl,y,rd=1246.76kNm, Vpl,z,rd=0.00kN

Ned=450.06kN <= 0.25x5538.00=0.25xNplrd=1384.50kN

Ned=450.06kN <=  $[10^{-3}] \times 0.5 \times 562.0 \times 12.0 \times 355 / 1.00 = 0.5 \text{ hw} \cdot \text{tw} \cdot f_y / \gamma_{M0} = 1197.06 \text{ kN}$

Effect of axial force is neglected

(EC3 §6.2.9.1 Eq.6.33, Eq.6.34, Eq.6.35)

Ved=0 kN, Effect of shear force is neglected

(EC3 §6.2.8.2)

My,ed= 89.32 kNm < 1246.76 kNm =Mply,rd, Check is verified

### Steel design, element 1, [Left end ], L= 4.600m, IPE 600

MedA= 78.05 kNm, VedA= 51.98 kN, NedA= -455.38 kN

#### Ultimate Limit State , Verification for compression (EN1993-1-1, §6.2.4)

Nc.ed=455.38 kN

Compression Resistance  $N_{plrd}=A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 15600 \times 355 / 1.00 = 5538.00 \text{ kN}$   
Ned= 455.38 kN < 5538.00 kN =Nc,rd=Nplrd, Check is verified

#### Ultimate Limit State , Verification for bending moment y-y (EN1993-1-1, §6.2.5)

My.ed= 78.05 kNm

Bending Resistance  $M_{ply,rd}=W_{ply} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 3512.0 \times 10^3 \times 355 / 1.00 = 1246.76 \text{ kNm}$   
My,ed= 78.05 kNm < 1246.76 kNm =My,rd=Mply,rd, Check is verified

#### Ultimate Limit State , Verification for axial force, shear and bending (EN1993-1-1, §6.2.9)

N.ed= 455.38kN (Compression), Vz.ed= 51.98kNm, My.ed= 78.05kN

Nplrd=5538.00kN, Mpl,y,rd=1246.76kNm, Vpl,z,rd=0.00kN

Ned=455.38kN <= 0.25x5538.00=0.25xNplrd=1384.50kN

Ned=455.38kN <=  $[10^{-3}] \times 0.5 \times 562.0 \times 12.0 \times 355 / 1.00 = 0.5 \text{ hw} \cdot \text{tw} \cdot f_y / \gamma_{M0} = 1197.06 \text{ kN}$

Effect of axial force is neglected

(EC3 §6.2.9.1 Eq.6.33, Eq.6.34, Eq.6.35)

Ved=0 kN, Effect of shear force is neglected

(EC3 §6.2.8.2)

My,ed= 78.05 kNm < 1246.76 kNm =Mply,rd, Check is verified

### Steel design, element 1, [Right end], L= 4.600m, IPE 600

MedB= -161.05 kNm, VedB= 51.98 kN, NedB= -447.77 kN

#### Ultimate Limit State , Verification for compression (EN1993-1-1, §6.2.4)

Nc.ed=447.77 kN

Compression Resistance  $N_{plrd}=A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 15600 \times 355 / 1.00 = 5538.00 \text{ kN}$   
Ned= 447.77 kN < 5538.00 kN =Nc,rd=Nplrd, Check is verified

#### Ultimate Limit State , Verification for bending moment y-y (EN1993-1-1, §6.2.5)

My.ed=161.05 kNm

Bending Resistance  $M_{ply,rd}=W_{ply} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 3512.0 \times 10^3 \times 355 / 1.00 = 1246.76 \text{ kNm}$   
My,ed= 161.05 kNm < 1246.76 kNm =My,rd=Mply,rd, Check is verified

**Ultimate Limit State , Verification for axial force, shear and bending**

(EN1993-1-1, §6.2.9)

N.ed= 447.77kN (Compression), Vz.ed= 51.98kNm, My.ed= 161.05kN

Nplrd=5538.00kN, Mpl,y,rd=1246.76kNm, Vpl,z,rd=0.00kN

Ned=447.77kN <= 0.25x5538.00=0.25xNplrd=1384.50kN

Ned=447.77kN <= [10<sup>-3</sup>]x0.5x562.0x12.0x355/1.00=0.5hw·tw·fy/γM0=1197.06 kN

Effect of axial force is neglected

(EC3 §6.2.9.1 Eq.6.33, Eq.6.34, Eq.6.35)

Ved=0 kN, Effect of shear force is neglected

(EC3 §6.2.8.2)

My,ed= 161.05 kNm < 1246.76 kNm =Mply,rd, Check is verified

**Steel design, element 1 , L= 4.600m, IPE 600, Buckling resistance**

**Flexural Buckling, (Ultimate Limit State )**

(EN1993-1-1, §6.3.1)

Buckling lengths: Lcr,y=1.870x4600=8600mm, Lcr,z=1.000x4600=4600mm

Non-dimensional slenderness (Cross-section Class: 1 )

(EC3 §6.3.1.3)

$\bar{\lambda}_y = \sqrt{(A \cdot f_y / N_{cr,y})} = (L_{cr,y} / i_y) \cdot (1 / \lambda_1) = (8600 / 243.0) \times (1 / 76.06) = 0.465$

$\bar{\lambda}_z = \sqrt{(A \cdot f_y / N_{cr,z})} = (L_{cr,z} / i_z) \cdot (1 / \lambda_1) = (4600 / 46.6) \times (1 / 76.06) = 1.298$

$\lambda_1 = \pi \sqrt{(E / f_y)} = 93.9$ ,  $\epsilon = \sqrt{(235 / f_y)} = 0.81$

h/b=600/220=2.73>=1.20, tf=19.0mm<=40 mm

y-y buclng curve:a, imperfection factor:αy=0.21, χy=0.935

(EC3 T.6.2,T.6.1, Fig.6.4)

$\Phi_y = 0.5 [1 + \alpha_y (\bar{\lambda}_y - 0.2) + \bar{\lambda}_y^2] = 0.5 \times [1 + 0.21 \times (0.465 - 0.2) + 0.465^2] = 0.636$

$\chi_y = 1 / [\Phi_y + \sqrt{(\Phi_y^2 - \bar{\lambda}_y^2)}] = 1 / [0.636 + \sqrt{(0.636^2 - 0.465^2)}] = 0.935 <= 1$  χy=0.935

z-z buclng curve:b, imperfection factor:αz=0.34, χz=0.428

$\Phi_z = 0.5 [1 + \alpha_z (\bar{\lambda}_z - 0.2) + \bar{\lambda}_z^2] = 0.5 \times [1 + 0.34 \times (1.298 - 0.2) + 1.298^2] = 1.529$

$\chi_z = 1 / [\Phi_z + \sqrt{(\Phi_z^2 - \bar{\lambda}_z^2)}] = 1 / [1.529 + \sqrt{(1.529^2 - 1.298^2)}] = 0.428 <= 1$  χz=0.428

Reduction factor  $\chi = 1 / [\Phi + \sqrt{(\Phi^2 - \bar{\lambda}^2)}]$ , χ<=1.0,  $\Phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2]$ , χ=0.428

(EC3 Eq.6.49)

Nb,rd=χ·A·fy/γM1= 0.428x[10<sup>-3</sup>]x15600x355/1.00=2370.26kN

(EC3 Eq.6.47)

Nc,ed= 455.38 kN < 2370.26 kN =Nb,rd, Check is verified

**Lateral torsional buckling**

(EN1993-1-1, §6.3.2)

Elastic critical moment for lateral-torsional buckling

(EC3 §6.3.2.2.2, EN1993:2002 AnnexC)

Timoshenko,S.P, Gere,J.M, Theory of elastic stability, McGraw-Hill, 1961

$M_{cr} = C_1 \cdot [\pi^2 EI_z / (kL)^2] \{ \sqrt{[(kz/kw)^2 (I_w / I_z) + (kL)^2 GI_t / (\pi^2 EI_z) + (C_2 \cdot z_g - C_3 \cdot z_j)^2]} - (C_2 \cdot z_g - C_3 \cdot z_j) \}$

$G = E / (2(1+\nu)) = 210000 / (2(1+0.30)) = 80769 = 8.1 \times 10^4$  N/mm<sup>2</sup>

k·L=4600mm, zg=0mm, zj=0mm

(EN1993:2002 T.C.1)

kz=1.0, kw=1.0, C1=2.567, C2=0.553, C3=0.000

(EN1993:2002 T.C.1)

$M_{cr} = [10^{-6}] 2.567 \times [\pi^2 \times 2.1 \times 10^5 \times 33.870 \times 10^6 / 4600^2]$

$\times \{ [1.0 \times (2845.5 \times 10^9 / 33.870 \times 10^6)]$

$+ 4600^2 \times 8.1 \times 10^4 \times 1.654 \times 10^6 / (\pi^2 \times 2.1 \times 10^5 \times 33.870 \times 10^6)]^{0.5} \} = 3002.8$  kNm

$\bar{\lambda}_{lt} = \sqrt{(W_{pl,y} \cdot f_y / M_{cr})} = \sqrt{[10^{-6}] \times 3512.0 \times 10^3 \times 355 / 3002.8} = 0.644$

(EC3 Eq.6.56)

h/b=600/220=2.73>2.00 buclng curve:b

imperfection factor:αlt=0.34, χlt=0.814

(EC3 T.6.2,T.6.1, Fig.6.4)

$\Phi_{lt} = 0.5 [1 + \alpha_{lt} (\bar{\lambda}_{lt} - 0.2) + \bar{\lambda}_{lt}^2] = 0.5 \times [1 + 0.34 \times (0.644 - 0.2) + 0.644^2] = 0.783$

$\chi_{lt} = 1 / [\Phi_{lt} + \sqrt{(\Phi_{lt}^2 - \bar{\lambda}_{lt}^2)}] = 1 / [0.783 + \sqrt{(0.783^2 - 0.644^2)}] = 0.814 <= 1$  χlt=0.814

Reduction factor  $\chi_{lt} = 1 / [\Phi_{lt} + \sqrt{(\Phi_{lt}^2 - \bar{\lambda}_{lt}^2)}]$ , χlt<=1.0, χlt=0.814

(Eq.6.56)

Mb,rd=χlt·Wpl,y·fy/γM1= 0.814x[10<sup>-6</sup>]x3512.0x10<sup>3</sup>x355/1.00=1014.86kNm

(EC3 Eq.6.55)

My,ed= 161.05 kNm < 1014.86 kNm =Mb,rd, Check is verified

**Axial force and bending moment,**

(EN1993-1-1, §6.3.3)

$N_{ed} / (\chi_y \cdot N_{rk} / \gamma_{M1}) + k_{yy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) <= 1$

(EC3 Eq.6.61)

$N_{ed} / (\chi_z \cdot N_{rk} / \gamma_{M1}) + k_{zy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) <= 1$

(EC3 Eq.6.62)

$N_{rk} = A \cdot f_y = [10^{-3}] \times 15600 \times 355 = 5538.0$  kN

(Tab.6.7)

$M_{y,rk} = W_{pl,y} \cdot f_y = [10^{-6}] \times 3512.0 \times 10^3 \times 355 = 1246.8$  kNm

$\chi_y \cdot N_{rk} / \gamma_{M1} = \chi_y \cdot A \cdot f_y / \gamma_{M1} = 0.935 \times [10^{-3}] \times 15600 \times 355 / 1.00 = 5178.0$  kN

$\chi_z \cdot N_{rk} / \gamma_{M1} = \chi_z \cdot A \cdot f_y / \gamma_{M1} = 0.428 \times [10^{-3}] \times 15600 \times 355 / 1.00 = 2370.3$  kN

$\chi_{LT} \cdot M_{y,rk} / \gamma_{M1} = \chi_{LT} \cdot W_{pl,y} \cdot f_y / \gamma_{M1} = 0.814 \times [10^{-6}] \times 3512.0 \times 10^3 \times 355 / 1.00 = 1014.9$  kNm

**Interaction factors, Method of computation: Method 2 Annex B**

(EC3 AnnexB)

$$k_{yy} = C_{my} [1 + (\bar{\lambda}_y - 0.2) \frac{N_{ed}}{(\chi_y \cdot N_{rk} / \gamma_{M1})}], \quad K_{yy} \leq C_{my} [1 + 0.8 \frac{N_{ed}}{(\chi_y \cdot N_{rk} / \gamma_{M1})}] \quad (\text{EC3 Tab.B.1})$$

$$k_{zy} = 1 - [0.10 \bar{\lambda}_z / (C_{m1t} - 0.25)] \frac{N_{ed}}{(\chi_z \cdot N_{rk} / \gamma_{M1})}, \quad K_{zy} \geq 1 - [0.10 / (C_{m1t} - 0.25)] \frac{N_{ed}}{(\chi_z \cdot N_{rk} / \gamma_{M1})}$$

$$\psi = -0.48, \quad C_{my} = 0.410, \quad C_{m1t} = 0.41,$$

$$K_{yy} = 0.41 \times [1 + (0.465 - 0.2) \times (455.4 / 5178.0)], \quad K_{yy} \leq 0.41 \times [1 + 0.8 \times (455.4 / 5178.0)]$$

$$K_{yy} = 0.420, \quad K_{yy} \leq 0.439, \quad K_{yy} = 0.420$$

$$K_{zy} = [1 - [0.10 \times 1.298 / (0.41 - 0.25)] \times (455.4 / 2370.3)], \quad K_{zy} \geq [1 - [0.10 / (0.41 - 0.25)] \times (455.4 / 2370.3)]$$

$$K_{zy} = 0.844, \quad K_{zy} \geq 0.880, \quad K_{zy} = 0.880$$

$$\frac{N_{ed}}{(\chi_y \cdot N_{rk} / \gamma_{M1})} + k_{yy} \cdot \frac{M_{y,ed}}{(\chi_{LT} \cdot M_{y,ed} / \gamma_{M1})} = \quad (\text{EC3 Eq.6.61})$$

$$455.4 / (0.935 \times 5538.0 / 1.00) + 0.420 \times 161.0 / (0.814 \times 1246.8 / 1.00) = 0.155$$

$$0.155 < 1.000, \quad \text{Check is verified}$$

$$\frac{N_{ed}}{(\chi_z \cdot N_{rk} / \gamma_{M1})} + k_{zy} \cdot \frac{M_{y,ed}}{(\chi_{LT} \cdot M_{y,ed} / \gamma_{M1})} = \quad (\text{EC3 Eq.6.62})$$

$$455.4 / (0.428 \times 5538.0 / 1.00) + 0.880 \times 161.0 / (0.814 \times 1246.8 / 1.00) = 0.332$$

$$0.332 < 1.000, \quad \text{Check is verified}$$

**Steel design, element 2 , L= 8.400m, IPE 500**

**Classification of cross-sections, Bending and compression**

(EN1993-1-1, §5.5)

Maximum and minimum cross-section stresses  $\sigma = N_{ed}/A_{el} \pm M_{y,ed}/W_{el,y} \pm M_{z,ed}/W_{el,z}$   
 $\sigma_1 = 88 \text{ N/mm}^2, \quad \sigma_2 = -79 \text{ N/mm}^2$  (compression positive)

Web

$c = 500.0 - 2 \times 16.0 - 2 \times 21.0 = 426.0 \text{ mm}, \quad t = 10.2 \text{ mm}, \quad c/t = 426.0 / 10.2 = 41.76$   
 S 355 ,  $t = 10.2 \leq 40 \text{ mm}, \quad f_y = 355 \text{ N/mm}^2, \quad \epsilon = (235/355)^{0.5} = 0.81$   
 Position of neutral axis for combined Bending and compression  
 $N_{ed} / (2t \cdot f_y / \gamma_{M0}) = 51978 / (2 \times 10.2 \times 355 / 1.00) = 7.2 \text{ mm}$   
 $\alpha = (426.0 / 2 + 7.2) / 426.0 = 0.517 > 0.5$   
 $c/t = 41.76 \leq 396 \times 0.81 / (13 \times 0.517 - 1) = 56.09$   
 The web is class 1 (EN1993-1-1, Tab.5.2)

Flange

$c = 200.0 / 2 - 10.2 / 2 - 21.0 = 73.9 \text{ mm}, \quad t = 16.0 \text{ mm}, \quad c/t = 73.9 / 16.0 = 4.62$   
 S 355 ,  $t = 16.0 \leq 40 \text{ mm}, \quad f_y = 355 \text{ N/mm}^2, \quad \epsilon = (235/355)^{0.5} = 0.81$   
 $c/t = 4.62 \leq 9 \epsilon = 9 \times 0.81 = 7.29$   
 The flange is class 1 (EN1993-1-1, Tab.5.2)

**Overall classification of cross-section is Class 1, Bending and compression  $N_{c,ed} + M_{y,ed}$**

**Resistance of cross-section**

(EN1993-1-1 §6.2.3, §6.2.4, §6.2.6, §6.2.5)

Tension Resistance  $N_{rdt}, r_d = A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 11550 \times 355 / 1.00 = 4100.25 \text{ kN}$   
 Compression Resistance  $N_{rdc}, r_d = A \cdot f_y / \gamma_{M0} = [10^{-3}] \times 11550 \times 355 / 1.00 = 4100.25 \text{ kN}$   
 Shear resistance  $V_{rdz}, r_d = A_v (f_y / \sqrt{3}) / \gamma_{M0} = [10^{-3}] \times 5985 \times (355 / 1.73) / 1.00 = 1226.72 \text{ kN}$   
 Bending Resistance  $M_{rdy}, r_d = W_{ely} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 778.87 \text{ kNm}$

**Steel design, element 2, [Span ], L= 8.400m, IPE 500**

**Med = 116.21 kNm, Ved = 0.00 kN, Ned = -51.98 kN**

**Ultimate Limit State , Verification for bending moment y-y**

(EN1993-1-1, §6.2.5)

**My,ed=116.21 kNm**

Bending Resistance  $M_{ply}, r_d = W_{ply} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 778.87 \text{ kNm}$   
 $M_{y,ed} = 116.21 \text{ kNm} < 778.87 \text{ kNm} = M_{y,rd} = M_{ply}, r_d, \quad \text{Check is verified}$



**Steel design, element 2, [Left end ], L= 8.400m, IPE 500**

MedA= -161.05 kNm, VedA= 132.02 kN, NedA= -51.98 kN

**Ultimate Limit State , Verification for bending moment y-y** (EN1993-1-1, §6.2.5)

My.ed=161.05 kNm

Bending Resistance  $M_{pl,y,rd} = W_{pl,y} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 778.87 \text{ kNm}$   
 My,ed= 161.05 kNm < 778.87 kNm =M<sub>y,rd</sub>=M<sub>pl,y,rd</sub>, Check is verified

**Ultimate Limit State , Verification for shear z** (EN1993-1-1, §6.2.6)

Vz.ed=132.02 kNm

$A_v = A - 2b \cdot t_f + (t_w + 2r) t_f = 11550 - 2 \times 200.0 \times 16.0 + (10.2 + 2 \times 21.0) \times 16.0 = 5985 \text{ mm}^2$  (EC3 §6.2.6.3)  
 $A_v = 5985 \text{ mm}^2 > \eta \cdot h_w \cdot t_w = 1.00 \times (500.0 - 2 \times 16.0) \times 10.2 = 1.00 \times 468.0 \times 10.2 = 4774 \text{ mm}^2$   
 Plastic Shear Resistance  $V_{pl,z,rd} = A_v (f_y / \sqrt{3}) / \gamma_{M0} = [10^{-3}] \times 5985 \times (355 / 1.73) / 1.00 = 1226.72 \text{ kN}$   
 Vz,ed= 132.02 kN < 1226.72 kN =V<sub>z,rd</sub>=V<sub>pl,z,rd</sub>, Check is verified

$h_w / t_w = (500.0 - 2 \times 16.0) / 10.2 = 468.0 / 10.2 = 45.88 \leq 72 \times 0.81 / 1.00 = 72 \epsilon / \eta = 58.32$  ( $\eta = 1.00$ )  
 S 355 ,  $t = 10.2 \leq 40$  mm,  $f_y = 355$  N/mm<sup>2</sup>,  $\epsilon = (235 / 355)^{0.5} = 0.81$   
 Shear buckling resistance is not necessary to be verified (EC3 §6.2.6.6)

**Ultimate Limit State , Verification for axial force, shear and bending** (EN1993-1-1, §6.2.9)

N.ed= 51.98kN (Compression), Vz.ed= 132.02kNm, My.ed= 161.05kN

$N_{pl,rd} = 0.00 \text{ kNm}$ ,  $M_{pl,y,rd} = 778.87 \text{ kNm}$ ,  $V_{pl,z,rd} = 1226.72 \text{ kN}$   
 Ned=0 kN, Effect of axial force is neglected (EC3 §6.2.9.1 Eq.6.33, Eq.6.34, Eq.6.35)  
 Ved=132.02kN <= 0.50x1226.72=0.50xV<sub>pl,z,rd</sub>=613.36kN  
 Effect of shear force is neglected (EC3 §6.2.8.2)  
 My,ed= 161.05 kNm < 778.87 kNm =M<sub>y,rd</sub>, Check is verified

**Steel design, element 2, [Right end], L= 8.400m, IPE 500**

MedB= -161.04 kNm, VedB= 132.02 kN, NedB= -51.98 kN

**Ultimate Limit State , Verification for bending moment y-y** (EN1993-1-1, §6.2.5)

My.ed=161.04 kNm

Bending Resistance  $M_{pl,y,rd} = W_{pl,y} \cdot f_y / \gamma_{M0} = [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 778.87 \text{ kNm}$   
 My,ed= 161.04 kNm < 778.87 kNm =M<sub>y,rd</sub>=M<sub>pl,y,rd</sub>, Check is verified

**Ultimate Limit State , Verification for shear z** (EN1993-1-1, §6.2.6)

Vz.ed=132.02 kNm

$A_v = A - 2b \cdot t_f + (t_w + 2r) t_f = 11550 - 2 \times 200.0 \times 16.0 + (10.2 + 2 \times 21.0) \times 16.0 = 5985 \text{ mm}^2$  (EC3 §6.2.6.3)  
 $A_v = 5985 \text{ mm}^2 > \eta \cdot h_w \cdot t_w = 1.00 \times (500.0 - 2 \times 16.0) \times 10.2 = 1.00 \times 468.0 \times 10.2 = 4774 \text{ mm}^2$   
 Plastic Shear Resistance  $V_{pl,z,rd} = A_v (f_y / \sqrt{3}) / \gamma_{M0} = [10^{-3}] \times 5985 \times (355 / 1.73) / 1.00 = 1226.72 \text{ kN}$   
 Vz,ed= 132.02 kN < 1226.72 kN =V<sub>z,rd</sub>=V<sub>pl,z,rd</sub>, Check is verified

$h_w / t_w = (500.0 - 2 \times 16.0) / 10.2 = 468.0 / 10.2 = 45.88 \leq 72 \times 0.81 / 1.00 = 72 \epsilon / \eta = 58.32$  ( $\eta = 1.00$ )  
 S 355 ,  $t = 10.2 \leq 40$  mm,  $f_y = 355$  N/mm<sup>2</sup>,  $\epsilon = (235 / 355)^{0.5} = 0.81$   
 Shear buckling resistance is not necessary to be verified (EC3 §6.2.6.6)

**Ultimate Limit State , Verification for axial force, shear and bending** (EN1993-1-1, §6.2.9)

N.ed= 51.98kN (Compression), Vz.ed= 132.02kNm, My.ed= 161.04kN

$N_{pl,rd} = 0.00 \text{ kNm}$ ,  $M_{pl,y,rd} = 778.87 \text{ kNm}$ ,  $V_{pl,z,rd} = 1226.72 \text{ kN}$   
 Ned=0 kN, Effect of axial force is neglected (EC3 §6.2.9.1 Eq.6.33, Eq.6.34, Eq.6.35)  
 Ved=132.02kN <= 0.50x1226.72=0.50xV<sub>pl,z,rd</sub>=613.36kN  
 Effect of shear force is neglected (EC3 §6.2.8.2)  
 My,ed= 161.04 kNm < 778.87 kNm =M<sub>y,rd</sub>, Check is verified

**Steel design, element 2 , L= 8.400m, IPE 500, Buckling resistance**

**Flexural Buckling, (Ultimate Limit State )**

(EN1993-1-1, §6.3.1)

Buckling lengths:  $L_{cr,y}=1.000 \times 8400=8400\text{mm}$ ,  $L_{cr,z}=0.250 \times 8400=2100\text{mm}$

Non-dimensional slenderness (Cross-section Class: 1 )

(EC3 §6.3.1.3)

$$\bar{\lambda}_y = \sqrt{(A \cdot f_y / N_{cr,y})} = (L_{cr,y} / i_y) \cdot (1 / \lambda_1) = (8400 / 204.3) \times (1 / 76.06) = 0.541$$

$$\bar{\lambda}_z = \sqrt{(A \cdot f_y / N_{cr,z})} = (L_{cr,z} / i_z) \cdot (1 / \lambda_1) = (2100 / 43.1) \times (1 / 76.06) = 0.641$$

$$\lambda_1 = \pi \sqrt{(E / f_y)} = 93.9 \text{e} = 76.06, \quad \varepsilon = \sqrt{(235 / f_y)} = 0.81$$

$h/b = 500 / 200 = 2.50 > 1.20$ ,  $t_f = 16.0\text{mm} \leq 40\text{ mm}$

y-y bucling curve:a, imperfection factor: $\alpha_y = 0.21$ ,  $\chi_y = 0.911$

(EC3 T.6.2, T.6.1, Fig.6.4)

$$\Phi_y = 0.5 [1 + \alpha_y (\bar{\lambda}_y - 0.2) + \bar{\lambda}_y^2] = 0.5 \times [1 + 0.21 \times (0.541 - 0.2) + 0.541^2] = 0.682$$

$$\chi_y = 1 / [\Phi_y + \sqrt{(\Phi_y^2 - \bar{\lambda}_y^2)}] = 1 / [0.682 + \sqrt{(0.682^2 - 0.541^2)}] = 0.911 \leq 1 \quad \chi_y = 0.911$$

z-z bucling curve:b, imperfection factor: $\alpha_z = 0.34$ ,  $\chi_z = 0.816$

$$\Phi_z = 0.5 [1 + \alpha_z (\bar{\lambda}_z - 0.2) + \bar{\lambda}_z^2] = 0.5 \times [1 + 0.34 \times (0.641 - 0.2) + 0.641^2] = 0.780$$

$$\chi_z = 1 / [\Phi_z + \sqrt{(\Phi_z^2 - \bar{\lambda}_z^2)}] = 1 / [0.780 + \sqrt{(0.780^2 - 0.641^2)}] = 0.816 \leq 1 \quad \chi_z = 0.816$$

Reduction factor  $\chi = 1 / [\Phi + \sqrt{(\Phi^2 - \bar{\lambda}^2)}]$ ,  $\chi \leq 1.0$ ,  $\Phi = 0.5 [1 + \alpha (\bar{\lambda} - 0.2) + \bar{\lambda}^2]$ ,  $\chi = 0.816$

(EC3 Eq.6.49)

$$N_{b,rd} = \chi \cdot A \cdot f_y / \gamma_{M1} = 0.816 \times [10^{-3}] \times 11550 \times 355 / 1.00 = 3345.80\text{kN}$$

(EC3 Eq.6.47)

$N_{c,ed} = 51.98\text{ kN} < 3345.80\text{ kN} = N_{b,rd}$ , Check is verified

**Lateral torsional buckling**

(EN1993-1-1, §6.3.2)

Elastic critical moment for lateral-torsional buckling

(EC3 §6.3.2.2.2, EN1993:2002 AnnexC)

*Timoshenko, S.P, Gere, J.M, Theory of elastic stability, McGraw-Hill, 1961*

$$M_{cr} = C_1 \cdot [\pi^2 E I_z / (k L)^2] \{ \sqrt{[(k z / k_w)^2 (I_w / I_z) + (k L)^2 G I_t / (\pi^2 E I_z) + (C_2 \cdot z_g - C_3 \cdot z_j)^2]} - (C_2 \cdot z_g - C_3 \cdot z_j) \}$$

$$G = E / (2(1+\nu)) = 210000 / (2(1+0.30)) = 80769 = 8.1 \times 10^4 \text{ N/mm}^2$$

$$k \cdot L = 2100\text{mm}, \quad z_g = h/2 = 500/2 = 250\text{mm}, \quad z_j = 0\text{mm}$$

(EN1993:2002 T.C.1)

$$k_z = 1.0, \quad k_w = 1.0, \quad C_1 = 1.623, \quad C_2 = 0.083, \quad C_3 = -2.587$$

(EN1993:2002 T.C.1)

$$M_{cr} = [10^{-6}] 1.623 \times [\pi^2 \times 2.1 \times 10^5 \times 21.420 \times 10^6 / 2100^2]$$

$$\times \{ [1.0 \times (1249.4 \times 10^9 / 21.420 \times 10^6)$$

$$+ 2100^2 \times 8.1 \times 10^4 \times 0.893 \times 10^6 / (\pi^2 \times 2.1 \times 10^5 \times 21.420 \times 10^6)$$

$$+ (0.083 \times 250)^2 \}^{0.5} - (0.083 \times 250) \} = 3856.0 \text{ kNm}$$

$$\bar{\lambda}_{lt} = \sqrt{(W_{pl,y} \cdot f_y / M_{cr})} = \sqrt{([10^{-6}] \times 2194.0 \times 10^3 \times 355 / 3856.0)} = 0.449$$

(EC3 Eq.6.56)

$h/b = 500 / 200 = 2.50 > 2.00$  bucling curve:b

imperfection factor: $\alpha_{lt} = 0.34$ ,  $\chi_{lt} = 0.906$

(EC3 T.6.2, T.6.1, Fig.6.4)

$$\Phi_{lt} = 0.5 [1 + \alpha_{lt} (\bar{\lambda}_{lt} - 0.2) + \bar{\lambda}_{lt}^2] = 0.5 \times [1 + 0.34 \times (0.449 - 0.2) + 0.449^2] = 0.643$$

$$\chi_{lt} = 1 / [\Phi_{lt} + \sqrt{(\Phi_{lt}^2 - \bar{\lambda}_{lt}^2)}] = 1 / [0.643 + \sqrt{(0.643^2 - 0.449^2)}] = 0.906 \leq 1 \quad \chi_{lt} = 0.906$$

Reduction factor  $\chi_{lt} = 1 / [\Phi_{lt} + \sqrt{(\Phi_{lt}^2 - \bar{\lambda}_{lt}^2)}]$ ,  $\chi_{lt} \leq 1.0$ ,  $\chi_{lt} = 0.906$

(Eq.6.56)

$$M_{b,rd} = \chi_{lt} \cdot W_{pl,y} \cdot f_y / \gamma_{M1} = 0.906 \times [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 705.66\text{kNm}$$

(EC3 Eq.6.55)

$M_{y,ed} = 161.05\text{ kNm} < 705.66\text{ kNm} = M_{b,rd}$ , Check is verified

**Axial force and bending moment,**

(EN1993-1-1, §6.3.3)

$$N_{ed} / (\chi_y \cdot N_{rk} / \gamma_{M1}) + k_{yy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) \leq 1$$

(EC3 Eq.6.61)

$$N_{ed} / (\chi_z \cdot N_{rk} / \gamma_{M1}) + k_{zy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) \leq 1$$

(EC3 Eq.6.62)

$$N_{rk} = A \cdot f_y = [10^{-3}] \times 11550 \times 355 = 4100.2 \text{ kN}$$

(Tab.6.7)

$$M_{y,rk} = W_{pl,y} \cdot f_y = [10^{-6}] \times 2194.0 \times 10^3 \times 355 = 778.9 \text{ kNm}$$

$$\chi_y \cdot N_{rk} / \gamma_{M1} = \chi_y \cdot A \cdot f_y / \gamma_{M1} = 0.911 \times [10^{-3}] \times 11550 \times 355 / 1.00 = 3735.3\text{kN}$$

$$\chi_z \cdot N_{rk} / \gamma_{M1} = \chi_z \cdot A \cdot f_y / \gamma_{M1} = 0.816 \times [10^{-3}] \times 11550 \times 355 / 1.00 = 3345.8\text{kN}$$

$$\chi_{LT} \cdot M_{y,rk} / \gamma_{M1} = \chi_{LT} \cdot W_{pl,y} \cdot f_y / \gamma_{M1} = 0.906 \times [10^{-6}] \times 2194.0 \times 10^3 \times 355 / 1.00 = 705.7\text{kNm}$$

**Interaction factors, Method of computation: Method 2 Annex B**

(EC3 AnnexB)

$$k_{yy} = C_{my} [1 + (\bar{\lambda}_y - 0.2) [N_{ed} / (\chi_y \cdot N_{rk} / \gamma_{M1})]], \quad K_{yy} \leq C_{my} [1 + 0.8 [N_{ed} / (\chi_y \cdot N_{rk} / \gamma_{M1})]]$$

(EC3 Tab.B.1)

$$k_{zy} = 1 - [0.10 \bar{\lambda}_z / (C_{m1t} - 0.25)] [N_{ed} / (\chi_z \cdot N_{rk} / \gamma_{M1})], \quad K_{yz} \geq 1 - [0.10 / (C_{m1t} - 0.25)] [N_{ed} / (\chi_z \cdot N_{rk} / \gamma_{M1})]$$

$$\psi = 1.00, \quad \alpha_s = 0.00, \quad C_{my} = 0.400, \quad C_{m1t} = 0.40,$$

$K_{yy}=0.40 \times [1 + (0.541 - 0.2) \times (52.0 / 3735.3)]$ ,  $K_{yy} \leq 0.40 \times [1 + 0.8 \times (52.0 / 3735.3)]$   
 $K_{yy} = 0.402$ ,  $K_{yy} \leq 0.404$ ,  $K_{yy} = 0.402$   
 $K_{zy} = [1 - [0.10 \times 0.641 / (0.40 - 0.25)] \times (52.0 / 3345.8)]$ ,  $K_{zy} \geq [1 - [0.10 / (0.40 - 0.25)] \times (52.0 / 3345.8)]$   
 $K_{zy} = 0.993$ ,  $K_{zy} \geq 0.990$ ,  $K_{zy} = 0.993$

$N_{ed} / (\chi_y \cdot N_{rk} / \gamma_{M1}) + k_{yy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) =$  (EC3 Eq.6.61)  
 $52.0 / (0.911 \times 4100.2 / 1.00) + 0.402 \times 161.0 / (0.906 \times 778.9 / 1.00) = 0.106$   
 $0.106 < 1.000$ , Check is verified

$N_{ed} / (\chi_z \cdot N_{rk} / \gamma_{M1}) + k_{zy} \cdot M_{y,ed} / (\chi_{LT} \cdot M_{y,rk} / \gamma_{M1}) =$  (EC3 Eq.6.62)  
 $52.0 / (0.816 \times 4100.2 / 1.00) + 0.993 \times 161.0 / (0.906 \times 778.9 / 1.00) = 0.242$   
 $0.242 < 1.000$ , Check is verified

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